

Multi-scale modelling of the trabecular bone elastoplastic behaviour under compression loading

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We propose a new elastoplastic damage coupled model for the modelling of trabecular bone behaviour. The damage is carried out thanks to the limit analysis based on the MCK criterion. We first present the methodology allowing the estimation of elastic anisotropic properties of porous media by means of Mori–Tanaka homogenisation scheme. Then, we present the formulation of the integrated yield criterion derived by considering trial velocity field inspired from the Eshelby inhomogeneous inclusion solution. The obtained micromechanical model is implemented via a UMAT routine within the explicit dynamic code LS-DYNA. The proposed micromechanical model has been applied successfully for the modelling of some biomechanics applications to estimate the mechanical properties of the bovine trabecular bone.

Nous proposons un nouveau modèle élastoplastique couplé à l'endommagement pour la modélisation du comportement mécanique de l'os trabéculaire. L'endommagement est pris en compte grâce à l'analyse limite basée sur le critère MCK. Nous présentons d'abord la méthodologie permettant l'estimation des propriétés élastiques anisotropes des milieux poreux par le schéma d'homogénéisation de Mori–Tanaka. Ensuite, nous présentons la formulation du critère d'écoulement intégré obtenu par un champ de vitesse test, inspiré de la solution d'inclusion Eshelby. Le modèle micromécanique est implémenté via une routine utilisateur UMAT dans le code dynamique explicite LS-DYNA. Le modèle micromécanique proposé est appliqué avec succès dans la modélisation de quelques applications en biomécanique pour estimer les propriétés mécaniques de l'os trabéculaire bovin.

Keywords: porous media; micromechanics; MCK criterion; trabecular bone; homogenisation.

Mots-clés: Milieux poreux; micromécanique; critère MCK; os trabéculaire; homogénéisation.

1. Introduction

The investigation of bone tissue as a material is an important topic in biomechanics. Motivated by clinical practice, a large amount of mechanical data has been collected for bovine and human bone with testing method conventionally used for engineering materials. After the introduction of artificial orthopaedic devices, the complex problem of bone implant interaction emerged and recalled the importance of the interrelations between biological and mechanical properties.

The mechanical behaviour of trabecular bone has been extensively studied in the literature as it is one of the main constituents of the bone tissue. Enclosed in compact bone at the end of epiphyses or in the core of flat and small bones, trabecular bone is subjected to multi-axial strains and stresses during physiological loading. Compressive mechanical testing represents

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an established method for evaluating the mechanical competence of trabecular bone, due to its simplicity and its resemblance to the natural deformation (Fyhrie & Schaffler, 1994). In compression, the mechanical behaviour of cortical and trabecular bone presents distinct characteristics. Cortical bone is quasi-brittle, while the mechanical behaviour of trabecular bone is typical of an elastoplastic cellular material (Gibson & Ashby, 1997; Hayes & Carter, 1976). For both types of bone under compressive monotonic loading, the tangent stiffness decreases already before the maximum force is reached. For trabecular bone, a smooth and gradual reduction of the stress (softening) then occurs until a local minimum, followed by rehardening (Gibson & Ashby, 1997). In studying this mechanical response, uniaxial mechanical compression tests were widely used to assess stiffness, strength, creep and fatigue properties of trabecular bone (Keaveny, Morgan, Niebur, & Yeh, 2001; Keaveny, Pinilla, Crawford, Kopperdahl, & Lou, 1997; Kosmopoulos, Schizas, & Keller, 2008; Rincón-Kohli, 2003). Especially, Kefalas and Eftaxiopoulos (2012) investigated experimentally the mechanical response of bovine trabecular bone subjected to uniaxial compression or tension under large strains was established. It has been revealed the three stages of the constitutive response, namely the initial increasing and softening branches at moderate strains, the plateau region at large strains and the hardening part at very large strains under compaction.

However, for many years, the differences in mechanical properties of trabecular bone were explained by volume fraction (Carter & Hayes, 1977; Rice, Cowin, & Bowman, 1988) and most of the relevant studies have been confined to one or few skeletal sites. It has been shown that the mechanical properties of trabecular bone are influenced by architectural anisotropy (Goulet et al., 1994; Hodgkinson & Currey, 1990). In order to describe the extent of anisotropy, the method of mean intercept length (MIL) has been introduced on planar sections (Whitehouse, 1974) and extended to 3D where its distribution was characterised by a second rank fabric tensor (Harrigan & Mann, 1984). In the pioneering work of Cowin (1985), a general relationship between volume fraction, fabric, which is a second-order tensor approximation of architectural anisotropy, and elastic properties was established.

From a modelling point of view, the relationship between morphology of spongy bones to the mechanical properties and failure mechanism can be assessed through computational means. The constitutive law, which defines the relation between stresses and strains, is incorporated in the FE method to model global behaviour. Most studies have used simple constitutive relations, for example linear elasticity with isotropic or transverse isotropic symmetry and the one-dimensional model of Fondrk, Bahniuk, and Davy (1999a) unsuitable for three-dimensional finite element simulations. So far, only little effort has been devoted to the development of inelastic constitutive models specific to bone (Fondrk, Bahniuk, and Davy, 1999b; Zysset & Curnier, 1996). In a consequent work, Gupta, Bayraktar, Fox, Keaveny, and Papadopoulos (2007) developed an infinitesimal plasticity-like model with isotropic kinematic hardening to describe yielding of trabecular bone at the continuum level. Based on the porous structure of the trabecular tissue, Gibson and Ashby (1997) have presented a multilinear parametric model up to compaction. In addition, Zysset and Curnier (1996) have proposed a three-dimensional constitutive law which describes anisotropic elasticity and the rate-independent degradation in mechanical properties from the growth of cracks or voids in the trabecular tissue. In Thurner et al. (2006), the authors have investigated the failure mechanism in trabecular bone specimens under compression. They have reported that trabecular bone often fails due to delamination, caused by microcracking and cracking, providing a mechanism for energy dissipation while conserving trabecular bone architecture. Recently, Charlebois, Jirásek, Philippe, and Zysset (2010) have proposed a constitutive law which captures the hardening–softening response of trabecular bone in compression and couples damage and plasticity with a non-local formulation (Grassl & Jirásek, 2006; Hansen & Schreyer, 1994; Lubarda & Krajcinovic, 1995; Maugin, 1992; Zysset & Curnier, 1996).

In this paper, we propose a micromechanical elastoplastic model of the bone behaviour in view of an integration of bone damage for the simulation of necking and rupture initiation. The determination of elastic properties has been made by using the Mori–Tanaka scheme and experimental measurements of anisotropy by microtomography techniques. The non-linear behaviour is performed by a coupling between plasticity and damage on a microscopic scale. The damage has been carried out by using a limit analysis approach based on the MCK criterion (Monchiet, Charkaluk Kondo, & Kondo, 2007, 2011). The derived yield function is performed by the consideration of new trial velocity fields inspired from the Eshelby (1957, 1959) exterior point solution to inclusions problem in which the eigen-strains are unknown. The obtained elastoplastic damage behaviour law is implemented via a user material routine UMAT within the explicit dynamic commercial code LS-DYNA (Hallquist, 2001) for the prediction of the global response of porous media. Finally, we present some applications in structural mechanics to study a cylindrical tensile specimen and in biological tissues to estimate the mechanical properties of the bovine trabecular bone. The adaptation of this model to biological tissues is made by considering a basic cell in conformity with the trabecular bone microstructure.

2. Formulation of the micromechanical model

2.1. Homogenisation scheme for elastic anisotropic properties

Micromechanical analysis provides links between macroscopic properties of material and its microstructure. In common homogenisation methods, the material properties are assumed to be homogeneous but unknown at macroscopic scale, whereas heterogeneous but known at microscopic scale. The main task is to find the homogeneous material properties at macroscopic scale based on the available information at microscopic scale. For this purpose, and as classically, a representative elementary volume (REV) is generally adopted to represent the idealised microstructure of material. In the present study, the REV is composed of an isotropic linear elastic solid matrix with elastic stiffness tensor $\mathbb{C}^{(m)}$ weakened by a random distribution of ellipsoidal shaped pores made up of phases r ($r = 1, \dots, N$) with the elasticity tensor $\mathbb{C}^{(r)}$. The inclusions of phase r are taken to be identical in shape and orientation with the normal \underline{n} , radius a and the average half-opening c .

Based on the matrix–inclusion problems (Eshelby, 1957; Laws & McLaughlin, 1979) and the Mori–Tanaka scheme (Mori & Tanaka, 1973), an estimate for the homogenised elastic stiffness tensor of the bone material reads as (Rahmoun, Chaari, Markiewicz, & Drazetic, 2009):

$$\mathbb{C}^{\text{hom}} = \mathbb{C}^{(m)} + \sum_{r=1}^N f^{(r)} (\mathbb{C}^{(r)} - \mathbb{C}^{(m)}) : \mathbb{A}_w^{(r)} : \left(f^{(m)} \mathbb{I} + \sum_{s=1}^N f^{(s)} \mathbb{A}_w^{(s)} \right)^{-1} \quad (1)$$

where $f^{(m)}$ and $f^{(r)}$ are, respectively, the volume fraction of matrix and inclusions and where the two sums are taken over all phases of the heterogeneous material in the REV. $\mathbb{A}_w^{(r)}$ denotes the strain concentration tensor associated with the r th phase family of inclusions of aspect ratio $w = \frac{c}{a}$, which writes:

$$\mathbb{A}_w^{(r)} = [\mathbb{I} + \mathbb{P}_w^{(r)} : (\mathbb{C}^{(r)} - \mathbb{C}^{(m)})]^{-1} \quad (2)$$

and $\mathbb{P}_w^{(r)}$ is the Hill tensor (Hill, 1965). This tensor depends on the shape and orientation of the r -th family of inclusions $\mathbb{C}^{(r)}$ (considered here as spheroid) and the elastic stiffness of the reference medium $\mathbb{C}^{(m)}$.

Finally, as in the previous work (Rahmoun et al., 2009), the formalism of stiffness tensor (1) is coupled with experimental measurements of architectural anisotropy obtained from X-ray micro tomography and the MIL method (Harrigan & Mann, 1984; Whitehouse, 1974), which will, in turn, be used to calculate the porosity.

The necessary detail can be found in Rahmoun et al. (2009), and only a summary of the main results of the model will be presented here.

2.2. The constitutive plastic-damage model based on the MCK criterion

In this section, we present an extension of the micromechanical linear model for the case of non-linear behaviour. The new model is based on a coupling between plasticity and damage on a microscopic scale. The damage has been carried out by using the framework of limit analysis of Gurson (1977) and its evolutions, in particular, the recent work of Monchiet et al. (2007, 2011), called MCK criterion. In this approach, the considered domain is composed of a spherical cavity with the radius a subjected to a uniform eigen-strain rate d^* and embedded in a spherical cell with the radius b . The spherical frame (coordinates r, φ, θ) is considered to describe the studied cell. The trial velocity field in the matrix, \underline{v} , required for the limit analysis of the hollow sphere (Gurson, 1977), classically takes the following general form:

$$\underline{v} = A.\underline{x} + \underline{v}^E \tag{3}$$

in which $A.\underline{x}$ is associated to a uniform strain rate A and \underline{v}^E is a heterogeneous field which corresponds to the expansion of the cavity. The matrix incompressibility implies that $tr(A) = 0$ and $div(\underline{v}^E) = 0$.

From the exterior-point Eshelby solution (Eshelby, 1959; Kachanov, Shafiro, Tsukrov, & Hanbook, 2003; Mura, 1987), a more refined velocity field \underline{v}^E , in the particular case of a spherical inclusion, can be given by:

$$\underline{v}^E = \sum_{r=1}^{r=6} \underline{v}^r d_r^* \tag{4}$$

where d_r^* is related to the eigen-strain in the inhomogeneity d^* . The microscopic plastic strain rate d derived from the velocity field (3) is defined by:

$$d = A + d^E \tag{5}$$

with $d^E = \nabla_s \underline{v}^E$, the symmetric part of the velocity gradient associated to \underline{v}^E (Monchiet et al., 2007, 2011). The microscopic dissipation $\pi(d) = d_0 d_{eq}$, where $d_{eq} = \sqrt{\frac{2}{3} d' : d'}$ is the equivalent plastic strain rate defined by (see Monchiet et al., 2007, 2011):

$$d_{eq}^2 = A_{eq}^2 + 2 \sum_{r=1}^{r=6} d_r^* A : d^r + \sum_{r=1}^{r=6} \sum_{s=1}^{s=6} d_r^* d_s^* d^r : d^s \tag{6}$$

and d' is the deviatoric part of strain tensor d . Then, in agreement with Hill–Mandel lemma, the macroscopic strain rate D is related to the local strain rate, d , by the average rule:

$$\frac{1}{|\Omega|} \int_{\Omega} ddV \tag{7}$$

where $|\Omega|$ is the volume of the studied cell (matrix + void).

The macroscopic plastic dissipation will be obtained after a minimisation procedure on the remaining unknown parameters d^{*} :

$$\Pi(D) = \min_{(d^*)'} [\tilde{\Pi}(D, (d^*)')] \quad \text{with} \quad \min_{(d^*)'} [\tilde{\Pi}(D, (d^*)')] = \frac{\sigma_0}{|\Omega|} \int_{\Omega-w} ddV \quad (8)$$

The yield surface, related the macroscopic dissipation, is performed by:

$$\Sigma = \frac{\partial \Pi}{\partial D} \quad (9)$$

Finally, following the limit analysis method of Gurson (1977), the approximate expression of the macroscopic yield function is obtained as (Monchiet et al., 2007, 2011):

$$\Phi(\Sigma, f) = \frac{\Sigma_{eq}^2}{\sigma_0^2} + 2f \cosh \left\{ \sqrt{\frac{9}{4} \frac{\Sigma_h^2}{\sigma_0^2} + \frac{2}{3} \frac{\Sigma_{eq}^2}{\sigma_0^2}} \right\} - 1 - f^2 = 0 \quad (10)$$

where Σ_h denotes the hydrostatic stress and f the material porosity. Σ_{eq} is the macroscopic von-Mises equivalent stress.

3. Numerical integration of the plastic-damage model

As stated before, we aim at formulating and implementing the MCK criterion. The implementation has been carried out within the explicit dynamic code LS-DYNA via the user-routine UMAT. For comparison purpose, the GTN model (which is a standard modification of the Gurson model) is also considered.

We introduce a hardening variable in the macroscopic yield function given by (10) by following a classical approach through the yield stress in the matrix $\bar{\sigma}$:

$$\Phi(\Sigma, \bar{\sigma}, f) = \frac{\Sigma_{eq}^2}{\bar{\sigma}^2} + 2f \cosh \left\{ \sqrt{\frac{9}{4} \frac{\Sigma_h^2}{\bar{\sigma}^2} + \frac{2}{3} \frac{\Sigma_{eq}^2}{\bar{\sigma}^2}} \right\} - 1 - f^2 = 0 \quad (11)$$

The macroscopic plastic flow rule obtained by upscaling, according to normality rule, is given by:

$$\dot{E}^p = \dot{\lambda} \frac{\partial \Phi}{\partial \Sigma} \quad (12)$$

in which $\dot{\lambda}$ is the plastic multiplier. Taking into account the incompressibility of the matrix, the porosity evolution law which characterises the damage growth, reads:

$$\dot{f} = 3(1 - f)\dot{E}_m^p + A\bar{\epsilon}_m^p \quad (13)$$

where E_m^p is the volumetric plastic strain and where $\bar{\epsilon}_m^p$ is the average volumetric plastic strain in the matrix. The therm A reads:

$$A = \frac{f_N}{s_N \sqrt{2\pi}} \exp \left\{ -\frac{1}{2} \left(\frac{\bar{\epsilon}_m^p - \epsilon_N}{s_N} \right)^2 \right\} \quad (14)$$

in which ε_N is the mean normal distribution of the nucleation strain, s_N is a standard deviation and f_N is the nucleates voids volume fraction. Therefore, the evolution of the yield stress in the matrix is given by (see for instance Gurson, 1977):

$$(1 - f)\bar{\sigma}\dot{\varepsilon} = \Sigma : \dot{E}^p \quad (15)$$

Finally, we adopt this simple coalescence model described by Tvergaard and Needleman (1984) in order to consider the effect of void coalescence in the numerical implementation:

$$f^* = \begin{cases} f & \text{if } f \leq f_c \\ f_c + \delta(f - f_c) & \text{if } f > f_c \end{cases} \quad (16)$$

where f_c represents the porosity at the onset of coalescence and $\delta = (f_U - f_c)(f_F - f_c)$ is a factor describing the acceleration of the material degradation during coalescence. f_F and f_U represent the porosity and the value of f^* at failure, respectively.

The stress increment calculation is based on the return mapping algorithm, which includes two steps.

First an elastic prediction of stress is done:

$$\Sigma_{n+1} = \mathbb{C}^{\text{hom}} : E_{n+1}^e = \mathbb{C}^{\text{hom}} : (E_{n+1} - E_{n+1}^p) = \Sigma^{\text{pred}} - \mathbb{C} : \Delta E^p \quad (17)$$

where \mathbb{C}^{hom} was determined by (1) and where Σ^{pred} is the elastic predictor expressed as: $\Sigma^{\text{pred}} = \mathbb{C}^{\text{hom}} : \Delta E$.

Then a plastic correction is used, we compute $\frac{\partial \Phi}{\partial \Sigma^{\text{pred}}}$ by using Equation (12) and we obtain the following expressions for the deviatoric part $(\Delta E^p)'$ and the hydrostatic part $\Delta \Sigma_m^p$:

$$\begin{cases} (\Delta E^p)' = \Delta \lambda \left(\frac{3}{\sigma^2} + \frac{2f}{\sigma^2} \frac{\sinh(R_0)}{R_0} \right) (\Delta \Sigma^{\text{pred}})' \\ \Delta \Sigma_m^p = \Delta \lambda \frac{3}{2} \frac{f \Sigma_m^{\text{pred}}}{\sigma^2} \frac{\sinh(R_0)}{R_0} \end{cases} \quad (18)$$

where $R_0 = \sqrt{\frac{9}{4} \frac{(\Sigma_m^{\text{pred}})^2}{\sigma^2} + \frac{2}{3} \frac{(\Sigma_{\text{eq}}^{\text{pred}})^2}{\sigma^2}}$.

Then, we have:

$$\Delta \lambda = \frac{\bar{\sigma}^2 \Delta E_{\text{eq}}^p}{\Sigma_{\text{eq}}^{\text{pred}}} \frac{1}{2 + \frac{4f}{3} \frac{\sinh(R_0)}{R_0}} \quad (19)$$

Finally, the consistency condition $\dot{\Phi} = 0$ implies to the expression of ΔE_{eq}^p and then to the plastic correction:

$$\Delta E^p = \Delta \lambda \frac{\partial \Phi}{\partial \Sigma^{\text{pred}}} \quad (20)$$

4. Finite element formulation

4.1. Kinematics

Now we will recall the formulation of the eight-node solid-shell element. With respect to nodal designation (Figure 1), the coordinate vector \mathbf{x} and the displacement vector \mathbf{u} of the element are (Hannachi, Naceur, & Batoz, 2007):

$$\mathbf{x} = \tilde{\mathbf{x}}(\xi, \eta) + \frac{\zeta}{2} \mathbf{V}(\xi, \eta) = \sum_{i=1}^4 N_i(\xi, \eta) \left(\frac{1-\zeta}{2} \mathbf{x}_i^- + \frac{1+\zeta}{2} \mathbf{x}_i^+ \right) \tag{21}$$

$$\mathbf{u} = \sum_{i=1}^4 N_i(\xi, \eta) \left(\frac{1-\zeta}{2} \mathbf{u}_i^- + \frac{1+\zeta}{2} \mathbf{u}_i^+ \right) \tag{22}$$

where $\tilde{\mathbf{x}}$ is the position vector at the mid-surface, \hat{h} and $\hat{\mathbf{n}}$ are the average thickness and the normal of the sheet, respectively, $N_i(\xi, \eta)$ are the two-dimensional eight-node Lagrangian interpolation functions, \mathbf{x}_i^- , \mathbf{u}_i^- and \mathbf{x}_i^+ , \mathbf{u}_i^+ are, respectively, the coordinate and displacement vectors of the i^{th} node on the bottom and top shell surfaces (Figure 1).

In this work, linear, isoparametric solid-shell elements are used with bilinear interpolation in membrane and linear interpolation in thickness direction. More details regarding the kinematics of solid-shell elements may be found in Hannachi et al. (2007).

4.2. Variational formulation

In order to deal with the several lockings separately, one needs to separate the expression of virtual internal work by uncoupling the membrane/bending, transverse/thickness and shearing (Hannachi et al., 2007).

$$W_{int} = \sum_{e=1}^{nelt} W_{int}^e; \quad W_{int}^e = W_{int}^{mb} + W_{int}^{tr} + W_{int}^{sh} \tag{23}$$

$$W_{int}^{mb} = \int_{V^0} \delta \mathbf{E}_{mb} \mathbf{S}_{mb} dV = \delta \mathbf{u}^T \mathbf{f}_{int}^{mb}; \quad \mathbf{f}_{int}^{mb} = \int_{V^0} \mathbf{B}_{mb}^T \mathbf{S}_{mb} dV \tag{24}$$

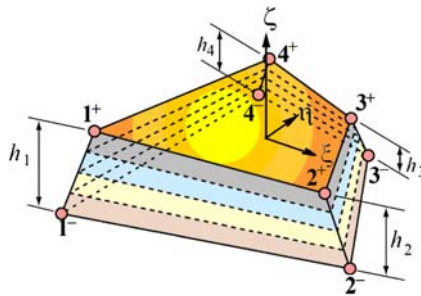


Figure 1. Eight-node solid-shell element.

$$W_{\text{int}}^{\text{tr}} = \int_{V^0} \delta \mathbf{E}_{\text{tr}} \mathbf{S}_{\text{tr}} dV = \delta \mathbf{u}^T \mathbf{f}_{\text{int}}^{\text{tr}}; \quad \mathbf{f}_{\text{int}}^{\text{tr}} = \int_{V^0} \mathbf{B}_{\text{tr}}^T \mathbf{S}_{\text{tr}} dV \quad (25)$$

$$W_{\text{int}}^{\text{sh}} = \int_{V^0} \delta \mathbf{E}_{\text{sh}} \mathbf{S}_{\text{sh}} dV = \delta \mathbf{u}^T \mathbf{f}_{\text{int}}^{\text{sh}}; \quad \mathbf{f}_{\text{int}}^{\text{sh}} = \int_{V^0} \mathbf{B}_{\text{sh}}^T \mathbf{S}_{\text{sh}} dV \quad (26)$$

with \mathbf{E} the Green–Lagrange strain tensor split into three components $\mathbf{E}_{\text{mb}} = \langle E_{11} E_{22} 2E_{12} \rangle$, $\mathbf{E}_{\text{tr}} = \langle E_{11} E_{22} E_{33} \rangle$ and $\mathbf{E}_{\text{sh}} = \langle 2E_{13} 2E_{23} \rangle$. The second Piola–Kirchhoff stress tensor \mathbf{S} is also decomposed into three parts $\mathbf{S}_{\text{mb}} = \langle S_{11} S_{22} S_{12} \rangle$, $\mathbf{S}_{\text{tr}} = \langle S_{11} S_{22} S_{33} \rangle$ and $\mathbf{S}_{\text{sh}} = \langle S_{13} S_{23} \rangle$. The internal force vector at the element level is then given by:

$$\mathbf{f}_{\text{int}}^{\text{u}} = \mathbf{f}_{\text{int}}^{\text{mb}} + \mathbf{f}_{\text{int}}^{\text{tr}} + \mathbf{f}_{\text{int}}^{\text{sh}} \quad (27)$$

A solid-shell element formulated using Equations (24–26) with standard integration based on a 2×2 Gauss schema in the in-plane of the shell element will fail because of numerous locking phenomena.

4.2.1. Remedies for shear locking

An effective method of resolving shear locking is the assumed natural strain (ANS) method in which the natural transverse shear strains are sampled and then interpolated at some discrete element points with a specific order.

The transverse shear strains $E_{\xi\zeta}$ and $E_{\eta\zeta}$ are calculated according to the average surface plan ($\zeta=0$), assuming that they vary linearly, and are function of E_{ξ} and E_{η} at the mid-side points (Figure 2):

$$E_{\xi\zeta}^{\text{ANS}} = \frac{1-\eta}{2} E_{\xi}^{\text{A1}} + \frac{1+\eta}{2} E_{\xi}^{\text{A2}}; \quad E_{\eta\zeta}^{\text{ANS}} = \frac{1-\zeta}{2} E_{\eta}^{\text{B1}} + \frac{1+\zeta}{2} E_{\eta}^{\text{B2}} \quad (28)$$

4.2.2. Remedies for trapezoidal locking

Similar to shear locking, trapezoidal locking occurs when lower order elements such as eight-node hexahedral elements are used to model curved shells, so that their cross sections assume the trapezoidal shape, these excessive number of sampled thickness strains can be reduced by

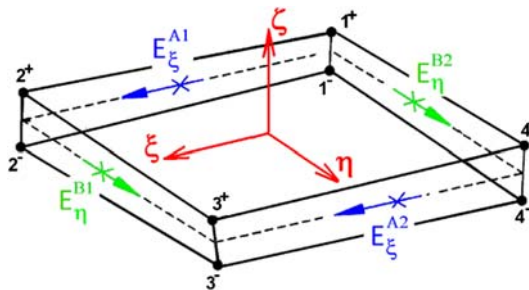


Figure 2. Shear locking treatment using ANS method.

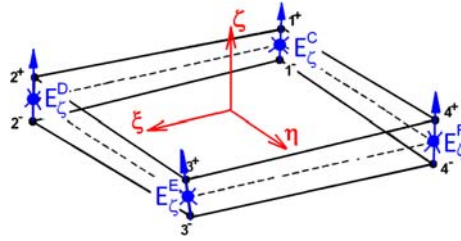


Figure 3. Trapezoidal locking treatment using ANS method.

using a bilinear interpolation of the transverse normal strains sampled at the four corners of the element mid-surface, namely (Figure 3)

$$E_{\zeta\zeta}^{ANS} = \sum_{i=1}^4 N_i(\zeta, \eta) E_{\zeta\zeta}(\zeta_i, \eta_i) \tag{29}$$

4.2.3. Remedies for volumetric locking

Material locking is controlled by a material parameter, the Poisson ratio ν .

Poisson’s ratio coupling requires the thickness strain to be a linear function of ζ . Because our solid-shell element has only two layers, as a consequence, the thickness strain does not vary with ζ ; thus, the element fail in reproducing the plane-stress condition.

$$\tilde{E}_{\zeta\zeta}^{EAS} = E_{\zeta\zeta}^{ANS} + \alpha \zeta t_{33} \tag{30}$$

where α represents the seventh independent internal parameter which will be eliminated by special condensation technique at the element level, t_{33} is required for transformation to the local element coordinates. An additional condition has now to be satisfied locally, leading to the increment of the additional degrees of freedom.

$$\frac{\partial W_{int}}{\partial \mathbf{u}} \Delta \mathbf{u} + \frac{\partial W_{int}}{\partial \alpha} \Delta \alpha = -W_{int}(\mathbf{u}, \alpha) \tag{31}$$

On an element level, the internal virtual work can now be computed with the compatible and the enhanced strains as in Equation (30).

$$\begin{aligned} W_{int} &= \sum_{i=1}^{nl} \int_{\xi} \int_{\eta} \int_{\zeta_i}^{\zeta_{i+1}} (\delta \mathbf{E}_{mb} \mathbf{S}_{mb} + \delta \mathbf{E}_{tr}^{EAS} \mathbf{S}_{tr} + \delta \mathbf{E}_{sh}^{ANS} \mathbf{S}_{sh}) J d\xi d\eta d\zeta \\ &= \delta \mathbf{u}^T \mathbf{f}_{int}^e \end{aligned} \tag{32}$$

nl is the number of layers and ζ_i is the transverse reference coordinate of the i th layer along the cross section of the element. The internal force vector \mathbf{f}_{int}^e can be obtained explicitly using compatible and enhanced strains:

$$\mathbf{f}_{int}^e = \mathbf{f}_{int}^u - \frac{f_\alpha}{k_{\alpha\alpha}} \mathbf{k}_{\alpha u} \tag{33}$$

where f_α is given by:

$$f_\alpha = \int_{V^0} \tilde{\mathbf{B}}_\alpha \mathbf{S} \, dV \tag{34}$$

where $\tilde{\mathbf{B}}_\alpha = \mathbf{T}_0 \mathbf{B}_\alpha$, $\mathbf{B}_\alpha = \langle 0 \ 0 \ \zeta \ 0 \ 0 \ 0 \rangle^T$ and \mathbf{T}_0 are the curvilinear transformation matrix.

The global internal force vector \mathbf{F} is then obtained by assembling element internal forces \mathbf{f}_{int}^e

$$\mathbf{F} = \sum_{e=1}^{nelt} \mathbf{f}_{int}^e \tag{35}$$

The use of a full integration in the in-plane of the shell element, through the use of 2×2 Gauss points, allows the obtention of a stiffness matrix that has a correct rank, this reveals that the element has no hourglass modes; therefore, stabilisation is not required in the present model.

5. Numerical applications

5.1. Validation of the model on necking prediction of a cylindrical tensile bar

For validation purpose only, we first present numerical predictions of the proposed MCK model on the necking of a cylindrical tensile bar. This example is considered as a standard benchmark since it has been studied extensively by several authors (Needleman, 1972; Tvergaard & Needleman, 1984) by using either Gurson or GTN models.

The cylindrical tensile bar of 8 cm length and 2 cm diameter is given in Figure 4. It is made of a common material with Young’s modulus $E = 300 \text{ N/cm}^2$ and a Poisson’s ratio $\nu = .3$. An isotropic hardening of the matrix has been considered through a power law of the form:

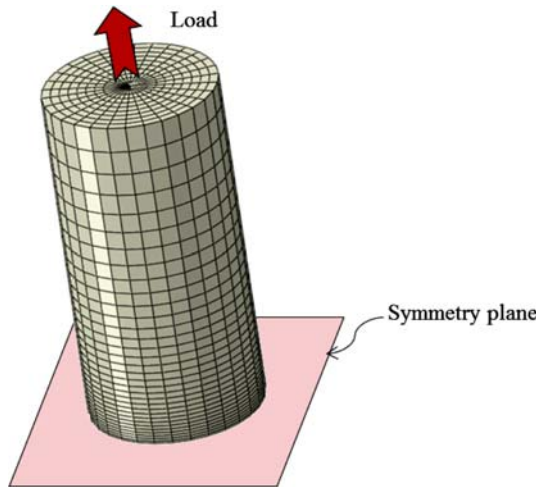


Figure 4. Finite element mesh of 1/2 cylindrical tensile test specimen.

$$\frac{\bar{\sigma}}{\sigma_0} = \left(\frac{\bar{\sigma}}{\sigma_0} + \frac{3G}{\sigma_0} \varepsilon_m^p \right)^N \tag{36}$$

where $\sigma_0 = 1$ is the initial yield stress, $N = .1$ is the hardening parameter and G is the elastic shear modulus.

As in Tveergard & Needleman (1984), the initial porosity $f_0 = 0$ and the coalescence parameters $f_F = .6$ and $f_c = .59$ have been adopted. The void nucleation parameters, $\varepsilon_N = .3$, $s_N = .1$ and $f_N = .04$, used are the same as those given by Needleman (1972). For comparison with GTN model, we consider parameters $q_1 = 1.5$, $q_2 = 1$ and $q_3 = 2.25$. These parameters were introduced by Tvergaard (1981) to make the predictions of the Guron model agree with numerical studies of an elastic-plastic medium containing a periodic array of voids.

The example problem focuses on the neck development, which is a precursor to failure in the form of cup–cone fracture. The formation of the neck results in a triaxial state of stress at the centre of the specimen, which accelerates the growth of the nucleated voids.

A finite element computation using our model implemented in LS-DYNA explicit dynamic code, using the 3D solid-shell mesh as illustrated in Figure 4. The specimen is fixed at the bottom edge and submitted to an imposed displacement to its top edge.

The predictions of nominal stress vs. the nominal strain of the present model and those obtained by the GTN model are given in Figure 5. It is shown that the maximum load is reached at a nominal strain of about 9%, after which the specimen softens due to coalescence of voids and eventually fractures across the neck region. Due to the explicit nature of our calculation, we observed a relative influence of the dynamic loading speed on the final results. Different loading velocities have been tested, and finally a velocity of .07 m/s is found to be a good compromise between efficiency and speed.

Moreover, it is observed that the results predicted by the present model recover completely those of GTN model. In fact, it is well known that the stress triaxiality in the centre of the cylindrical tensile specimen is very low.

Figures 6 and 7 show total void volume fraction and pressure stress contours, respectively, at 11% for the two previous models. It appears that, for the porosity and for low stress triaxiality, the present model and the GTN one are in good agreement.

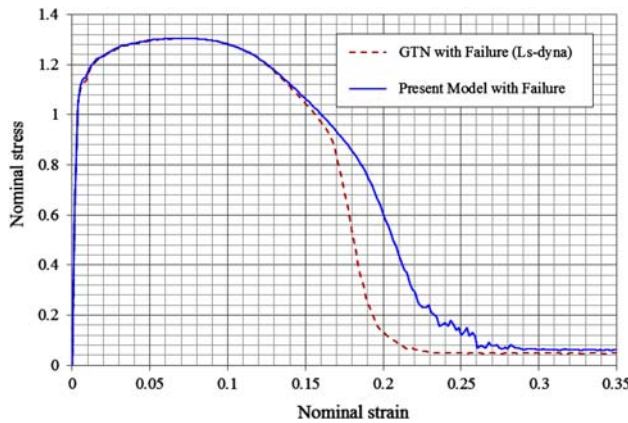


Figure 5. Stress–strain comparison between the present and the GTN models.

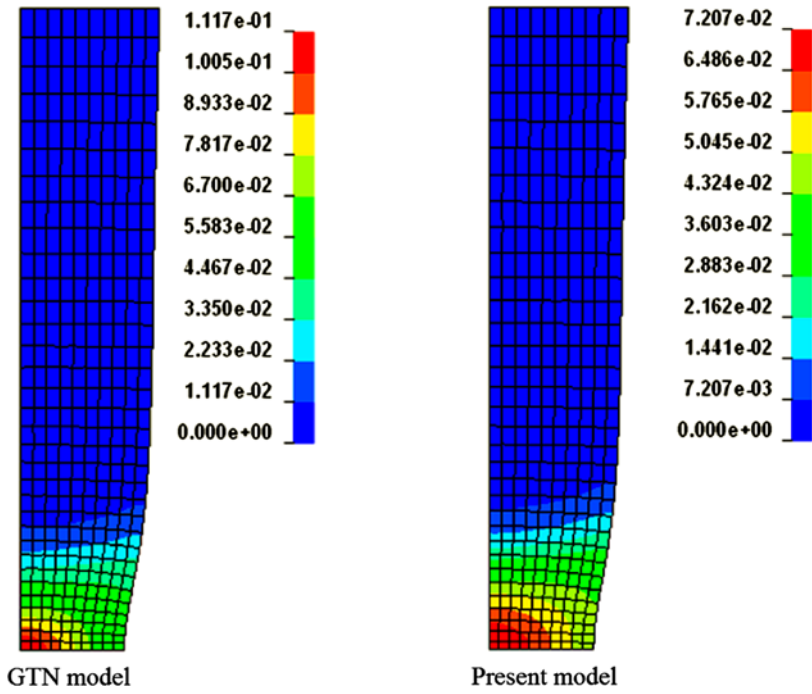


Figure 6. Void volume fraction distribution of the 1/4 specimen at time=9 ms.

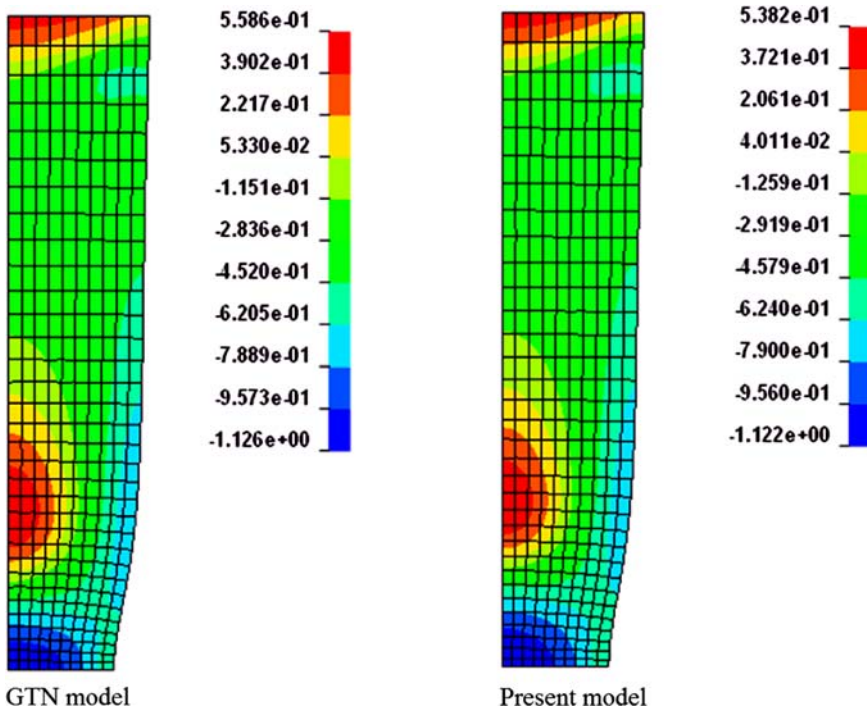


Figure 7. Hydrostatic pressure distribution of the 1/4 specimen at time=9 ms.

5.2. Modelling of trabecular bovine bone behaviour

This section is devoted to the adaptation of our elastoplastic damage model to biological tissues by considering a basic cell in conformity with the trabecular bone microstructure. For this end, an estimate for the mechanical properties of the bone trabeculae can be identified by inverse methods using the experimental data of bovine trabecular bone described in Halgrin (2009).

In their previous work, Halgrin, Rahmoun, Chaari, Markiewicz, & Drazetic, 2009 performed uniaxial compression tests of four cubic samples $10 \times 10 \times 10 \text{ mm}^3$ extracted from the same bovine rib at a constant loading velocity of 10 mm/min until the complete compaction of the sample.

Figure 8 shows the mechanical response of nominal stress against nominal strain for all four samples. We can observe that trabecular bone exhibits four different regimes, similar to those identified with other cellular materials (Gibson & Ashby, 1997). A linear elastic behaviour is reached at low compression strains. Then, a peak stress occurs at strains of 10%. After elastic buckling, the specimen seems to crush progressively with a quasi-constant stress. For strains higher than 60%, the total compaction of the sample occurs and the compression force increases quickly, this last phase is known to be *consolidation regime*.

We conducted an inverse identification procedure to identify the bone matrix material parameters. It consists of fitting computed results and experimental measurements by finding the optimal set of constitutive parameters (Young's modulus E_m and the yield stress of the bone matrix $\bar{\sigma}$). The volume void fraction is measured by microtomography for each sample and the aspect ratio of spherical voids is fixed to $w = 1$. The average value of volume void fraction based on experimental measurements using the microtomography is 77%. It is important to notice that in the present formulation, the REV is not modelled classically using very fine mesh as reported by Halgrin et al. (2009). Therefore, thanks to the Mori–Tanaka scheme, the REV is modelled using a simple mapped mesh of only 1000 solid-shell elements (see Figure 9).

Table 1 summaries the obtained results using an inverse identification procedure. As we can observe, the identified Young's modulus of the bovine bone matrix E_m shows slight variations from 1700 MPa to 2000 MPa, this is a direct consequence of the variations observed on the measurements data themselves. The same conclusions could be drawn, when observing the identified yield stress of the bone matrix $\bar{\sigma}$, which varies from 30 MPa to 40 MPa. Finally,

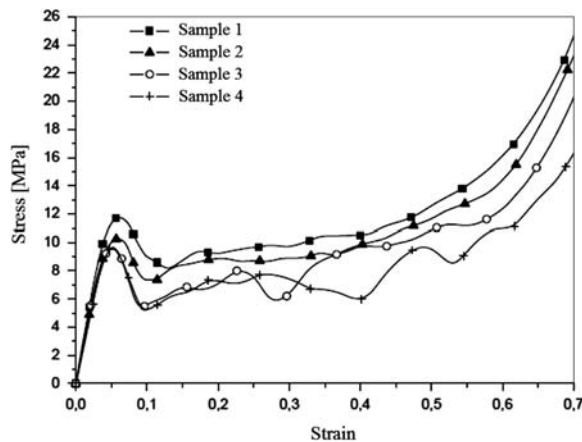


Figure 8. Experimental results of the four bovine rib samples under compression.

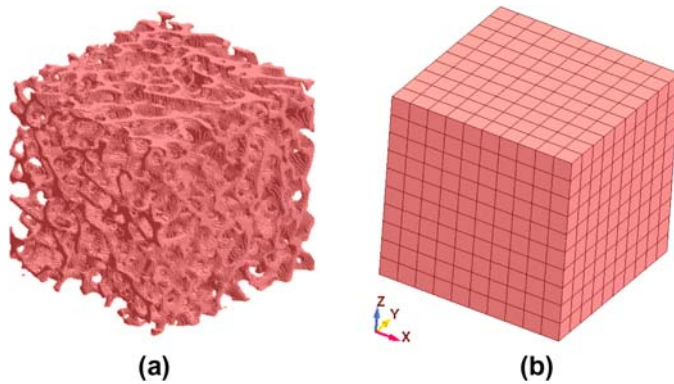


Figure 9. FE model: (a) REV classical fine mesh (424707 Hex elements in Halgrin et al., 2009) and (b) Present model (10 × 10 × 10 solid-shell elements).

Table 1. Trabecular bovine bone material parameters.

	Sample 1	Sample 2	Sample 3	Sample 4	Min–max
E_m (MPa)	2000	1700	1700	1700	1700–2000
$\bar{\sigma}$ (MPa)	40	38	33	30	30–40
E_{app} (MPa)	279.82	240.64	238.51	236.68	236.68–279.82

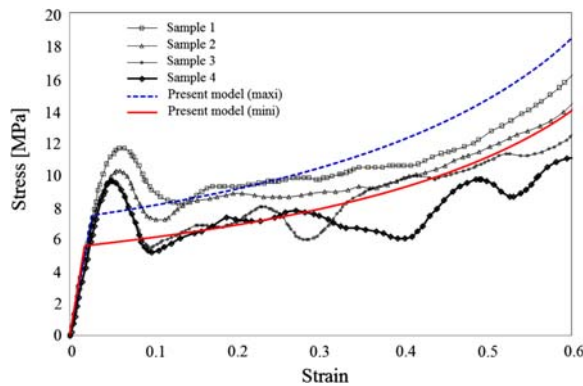


Figure 10. Numerical vs. experimental results of bovine trabecular bone behavior under compression.

thanks to the homogenised material tensor obtained, using the Mori–Tanaka scheme one can obtain the apparent Young’s modulus E_{app} , which is usually used in biomechanical applications. These results are also presented in Figure 10.

6. Conclusion

In conclusion, the framework of homogenisation was used to derive a yield criterion using an approximate limit-analysis based on the MCK criterion for porous materials. The elastic properties of these materials were determined by a coupling from the Mori–Tanaka scheme and experimental measurements of anisotropy using microtomography techniques. The

obtained model matches correctly the results obtained by the GTN model via a cylindrical tensile test specimen and allows to identify the mechanical response of trabecular bone. Its applications to the human femur bone are the subject of a forthcoming work.

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